

# ME451

# Kinematics and Dynamics of Machine Systems

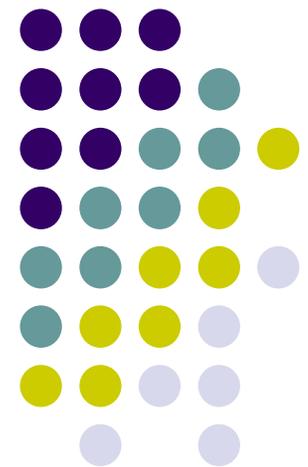
---

Dynamics of Planar Systems

December 6, 2011

Equilibrium Analysis & Inverse Dynamics Analysis

ME451 Wrap Up



# Before we get started...



- Last Time:
  - Finished the discussion on how to solve the Differential Algebraic Equations that govern the time evolution (dynamics) of a mechanism
    - Covered Newmark's method and discussed about the quasi-Newton approach that will help with the solution of the nonlinear system obtained after the discretization of the constrained equations of motion
- Today
  - Discuss the final exam
  - Inverse Dynamics Analysis
  - Equilibrium Analysis
  - We hug each other and stuff
- No more assignments in ME451 (MATLAB or textbook-based)
- Final Exam
  - Tu, Dec. 17 at 2:45 PM, Room: 1255ME

# Final Exam Related



- For simEngine2D, please make sure you support the following modeling elements:
  - Bodies
  - Abs constraints, revolute & translational joint, distance constraints
  - Point forces, Translational-Spring-Damper Forces, Rotational-Spring-Damper Torques
  - Computation of reaction forces in joints
- Note: only the last 15 points (out of 100) of the final exam are tied to simEngine2D. In other words, if you don't even have a simEngine2D and answer all the other questions right you'll end up with a score of 85.
- Do this prior to December 17:
  - Stop by 1255ME and check that your simEngine2D works as expected in conjunction with the version of MATLAB available on the computers in that room

# Final Exam, Rules of Engagement



- Identify the set of constraints present in the mechanism
- Determine a set of consistent Initial Conditions for the mechanism
- Define an acf and adm pair of files associated with the type of analysis that you are supposed to carry out and the model that you were given.
- Run simulations (Kinematics and/or Dynamics) using your simEngine2D.
- Generate a set of plots that show the time evolution of an attribute of the model (the motion of a point, the value of a reaction force as a function of time, etc.)
- Use the Learn@UW drop-box to provide a zipped directory that contains your code, adm/acf files, and png plots of your results. The naming convention for this directory should be “LastnameFinalME451.zip”. For instance, “NegrutFinalME451.zip”.

# Final Exam, Comments



- Running your `simEngine2D` code should also report the amount of time it took for completing a simulation. This information should be included in the zipped directory in a text file “`readme.txt`”.
- I will not insist on having `simEngine2D` that you use during the exam be implemented exclusively by you. However, in good faith, you will have to indicate in the text file “`readme.txt`” the percentage of your contribution to the `simEngine2D` code that you are using in the final exam. I will then understand that the remaining percent came from code written by other ME451 colleague[s]. This is absolutely fine (it won’t impact your final’s score), but should be acknowledged.
- If you contributed more than 66% to your `simEngine2D`, you qualify for entering the race for the fastest solver. Winning that race translates into an automatic A-grade in the course.
- One other automatic A grade *might* be assigned for the most general, flexible, and neatly organized `simEngine2D` code.

# Miscellaneous Tidbits...



- No more ME451 lectures from now till the final exam
  - Make-up class of last Th to count as lecture for this Th
  - Last week: work on your take-home midterm and on the Final Project
- I will travel so I won't be able to meet face-to-face with you. Post questions on the forum and I'll do my best to answer them as soon as possible
- Your homework with the lowest score will be dropped when computing the final grade

[New Topic]

# Inverse Dynamics: The idea



- First of all, what does dynamics analysis mean?
  - You apply some forces/torques on a mechanical system and look at how the configuration of the mechanism changes in time
  - How it moves also depends on the ICs associated with that mechanical system
- In \*inverse\* dynamics, the situation is quite the opposite:
  - You specify driving constraints on the mechanical system and you are interested in finding out the set of forces/torques that were actually applied to the mechanical system to lead to this motion
  - Note that you need a zero degree of freedom system after the application of the driving constraints
- When is \*inverse\* dynamics useful?
  - It's useful in controls. For instance in controlling the motion of a robot: you know how you want this robot to move, but you need to figure out what joint torques you should apply to make it move the way it should

# Inverse Dynamics: The Math



- When can one talk about Inverse Dynamics?
  - Given a mechanical system, a prerequisite for Inverse Dynamics is that the number of degrees of freedom associated with the system is **zero**
    - You have as many generalized coordinates as constraints (THIS IS KEY)
  - This effectively makes the problem a Kinematics problem
- The two stages of the Inverse Dynamics analysis
  - First solve for accelerations (recall the acceleration equation):

$$\Phi_{\mathbf{q}} \ddot{\mathbf{q}} = \gamma$$

- Next you solve for the Lagrange multipliers and then the reaction forces:

$$\Phi_{\mathbf{q}}^T \lambda = \mathbf{Q}^A - \mathbf{M} \ddot{\mathbf{q}}$$

# Inverse Dynamics: Closure



- Are we done once we computed the reaction forces?
  - Yes, because among the forces you computed, you get all the forces/torques that are necessary to impose the driving constraints  $\Phi^D$  that you imposed on the system

$$\mathbf{F}^D = - [\Phi_{\mathbf{r}_i}^D]^T \lambda$$

$$\mathbf{T}^D = [(\mathbf{s}'_i^P)^T \mathbf{B}_i^T [\Phi_{\mathbf{r}_i}^D]^T - [\Phi_{\phi_i}^D]^T] \lambda$$

Here constraint  $\Phi^D$  acts between body  $i$  and some other body. Reaction forces are computed as “felt” by body  $i$

- This gives you the forces/torques that you need to apply to get the prescribed motion



# **End Inverse Dynamics Beginning Equilibrium Analysis**

[New Topic]

# Equilibrium Analysis: The Idea



- A mechanical system is in equilibrium if the following conditions hold:

$$\dot{\mathbf{q}} = \mathbf{0} \quad \& \quad \ddot{\mathbf{q}} = \mathbf{0}$$

- Equivalently, the system is at rest, with zero acceleration
- So what does it take to be in this state of equilibrium?
  - You need to be in a certain configuration  $\mathbf{q}$
  - The reaction forces, that is, Lagrange Multipliers, should assume certain values
  - What does “certain” mean?

# Equilibrium Analysis: The Math



- Equations of Motion:

$$\mathbf{M}\ddot{\mathbf{q}} + \Phi_{\mathbf{q}}^T \lambda = \mathbf{Q}^A \quad \Rightarrow \quad \Phi_{\mathbf{q}}^T \lambda = \mathbf{Q}^A$$

- Position Constraint Equations:

$$\Phi(\mathbf{q}, t) = 0$$

- Velocity Constraint Equations:

$$\Phi_{\mathbf{q}} \cdot \dot{\mathbf{q}} = -\Phi_t \quad \Rightarrow \quad 0 = -\Phi_t$$

- Acceleration Constraint Equations:

$$\Phi_{\mathbf{q}} \ddot{\mathbf{q}} = -(\Phi_{\mathbf{q}} \dot{\mathbf{q}})_{\mathbf{q}} \dot{\mathbf{q}} - 2\Phi_{\mathbf{q}t} \dot{\mathbf{q}} - \Phi_{tt} \quad \Rightarrow \quad 0 = -\Phi_{tt}$$

[Cntd.]

# Equilibrium Analysis: The Math



- To conclude, one needs a configuration  $\mathbf{q}$  and the Lagrange multipliers  $\lambda$  should be such that

$$\Phi_{\mathbf{q}}^T \lambda = \mathbf{Q}^A$$

$$\Phi(\mathbf{q}, t) = \mathbf{0}$$

$$\Phi_t(\mathbf{q}, t) = \mathbf{0}$$

- How can you go about finding such a configuration?
  - Approach 1 (dumb, but powerful)
    - Add damping in a system and watch it move till it stops
  - Approach 2 (OK, but you need a good starting point)
    - Simply solve the nonlinear system to find  $\mathbf{q}$  and  $\mathbf{Q}^A$

$$\begin{aligned}\Phi_{\mathbf{q}}^T \lambda &= \mathbf{Q}^A \\ \Phi(\mathbf{q}) &= \mathbf{0}\end{aligned}$$

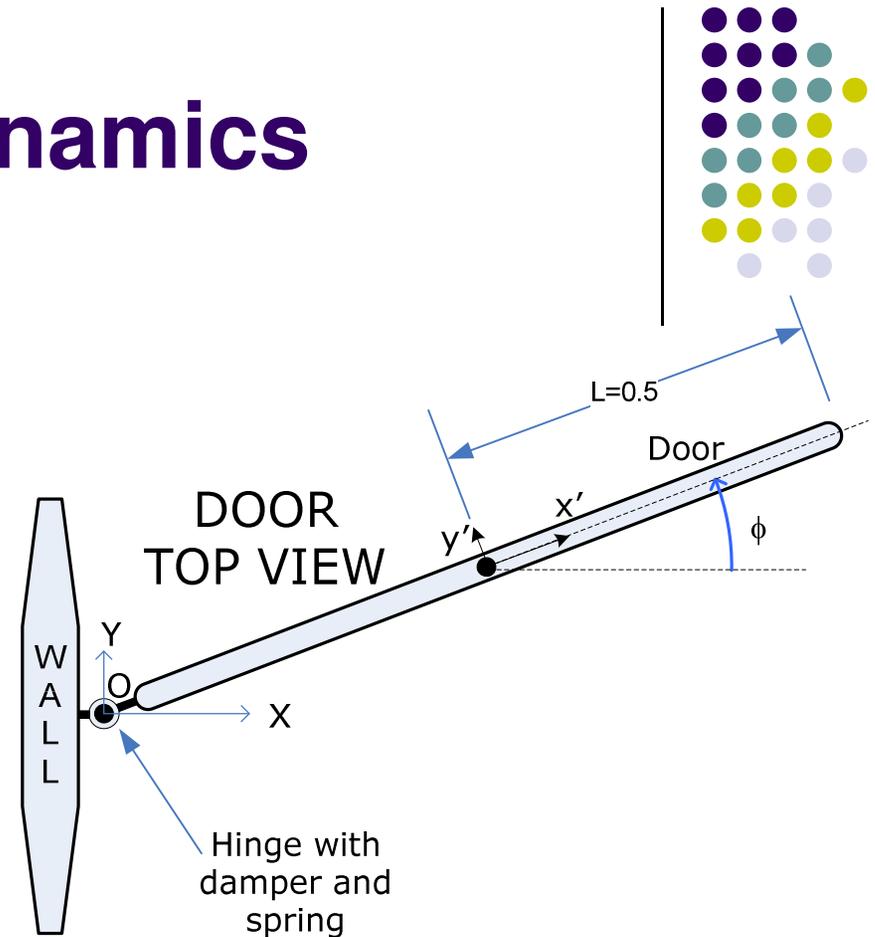
- Approach 3 (not that common)
  - Cast it as an optimization problem
  - Works for conservative systems only

[AO]

# Example: Inverse Dynamics

- Door Mass  $m = 30$
- Mass Moment of Inertia  $J' = 2.5$
- Spring/damping coefficients:  
 $K = 8$                        $C = 1$
- All units are SI.
- Zero Tension Angle of the spring:  
 $\phi_{free} = 0$
- Compute torque that electrical motor applies to *open* handicapped door
  - Apply motion for two seconds to open the door like

$$\Phi^D(t) = \phi - \frac{\pi}{2} \sin\left(\frac{\pi}{4}t\right)$$





[AO]

# Example: Equilibrium Analysis

- Find the equilibrium configuration of the pendulum below
  - Pendulum connected to ground through a revolute joint and rotational spring-damper element

- Free angle of the spring:

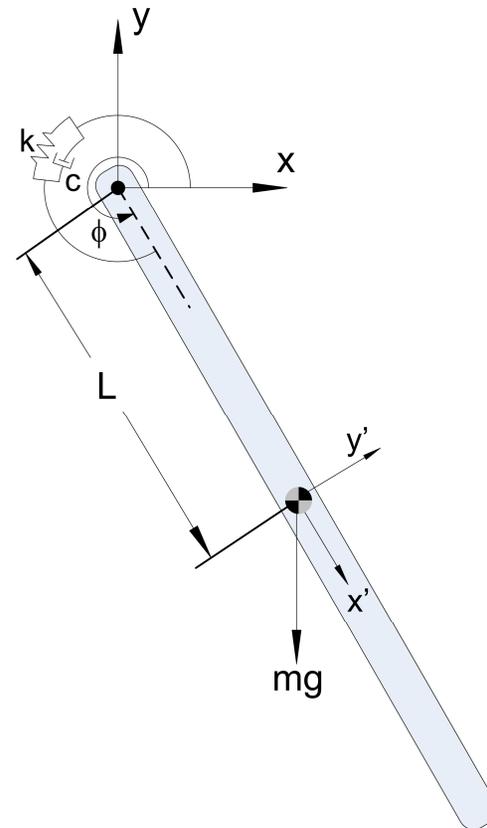
$$\phi_{free} = 0$$

- Spring constant:  $k=25$

- Mass  $m = 10$

- Length  $L=1$

- All units are SI.



# Summary of ME451



- Pick a set of Cartesian generalized coordinates  $q$ .
- You can then formulate the following sets of equations

- Equations of Motion: 
$$\mathbf{M}\ddot{\mathbf{q}} + \Phi_{\mathbf{q}}^T \lambda = \mathbf{Q}^A$$

- Position Constraint Equations: 
$$\Phi(\mathbf{q}, t) = \mathbf{0}$$

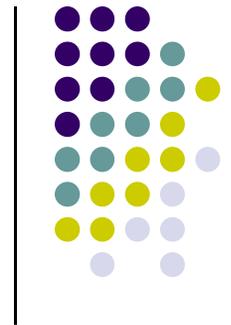
- Velocity Constraint Equations: 
$$\Phi_{\mathbf{q}} \dot{\mathbf{q}} = \nu$$

- Acceleration Constraint Equations: 
$$\Phi_{\mathbf{q}} \ddot{\mathbf{q}} = \gamma$$

# What was ME451 good for?



- Understand how to pose a Kinematics/Dynamics problem
- Understand how to solve a Kinematics/Dynamics problem
- Improve your MATLAB programming skills



**End ME451**